

AIR DUCT OUTLETS WITH MANUAL AND AUTOMATIC AIR STREAM DIRECTION CONTROL

RELATED APPLICATION

This application claims the benefit of U.S.
Provisional Application No. 60/444,335 filed January 31,
2003, the disclosure of which is incorporated herein by
5 reference in its entirety as if set forth fully herein.

FIELD OF THE INVENTION

The present invention relates generally to
vehicles and, more particularly, to air duct outlets
10 utilized within vehicles.

BACKGROUND OF THE INVENTION

Conventionally, vehicle interiors are provided
with one or more air duct outlets which are connected by
15 ducts to an outside air source and/or to a heating and/or
air conditioning system that provides cooled and/or
heated air. Because it is generally desirable for vehicle
occupants to be able to adjust the direction of air flow
within a vehicle interior, air duct outlets are typically
20 provided with adjustable vanes or louvers. In addition,
air duct outlets may be provided with dampers for
allowing vehicle occupants to control the amount of air
flowing therethrough.

Conventional air duct outlets utilize one or
25 more sets of louvers to mechanically redirect air
streams. While this technology is proven and successful,
the louver assemblies can be complex and expensive to

manufacture. In addition, conventional air duct outlets typically provide an air stream deflection range up to only about 90° (i.e., -45° to +45° relative to a centerline of an air duct outlet). Deflection ranges in excess of ±45° typically result in large pressure drops and lower air flow rates, which may be undesirable. In addition, vehicle manufacturers are continuously seeking components, such as air duct outlets, that have enhanced functionality and durability, yet are cost effective to manufacture.

Vehicle customers are also demanding air duct outlets that can provide automatic oscillation of air flow within a vehicle. As such, there is a need for air duct outlets that allow users to manually direct air flow and that also provide automatic oscillation of air flow.

SUMMARY OF THE INVENTION

In view of the above discussion, air duct outlets that provide automatic oscillation of air flow and that also allow users to manually redirect air streams within a wide range of directions are provided. According to embodiments of the present invention, an air duct outlet includes a housing having an air passageway that terminates at an air outlet. A reciprocating member is movably secured to the housing and is configured for reciprocal movement along a longitudinal axis thereof. A drive source is operably connected to the reciprocating member and transmits reciprocating motion to the reciprocating member. An air-directing vane is pivotally attached within the air passageway and is movable within a range of positions to direct an air stream flowing through the air passageway and exiting from the outlet. An actuator extends from the housing and is movable between first, second and third positions. The actuator is operably coupled with a drive arm that is operably coupled with the air-directing vane. A user can manually

5 pivot the air-directing vane within the range of
positions via movement of the actuator between the first
and second positions. When the actuator is moved to a
third position, the reciprocating member operably engages
the drive arm and causes the air-directing vane to
oscillate automatically within the range of positions.

10 Air duct outlets according to embodiments of
the present invention allow for improved air flow
directability including greatly increased ranges of
directability. Moreover, air duct outlets according to
embodiments of the present invention can facilitate new
and improved styling opportunities for vehicle
manufacturers and other manufacturers. Air duct outlets
according to embodiments of the present invention are not
15 limited to uses within vehicles (e.g., automobiles,
trucks, boats, aircraft, etc.), but may also be utilized
in residential and commercial heating, ventilating and
air conditioning (HVAC) applications.

20 BRIEF DESCRIPTION OF THE DRAWINGS

The accompanying drawings, which form a part of
the specification, illustrate key embodiments of the
present invention. The drawings and description together
serve to fully explain the invention.

25 **Fig. 1** is a front, perspective view of an air
duct outlet, according to embodiments of the present
invention.

Fig. 2 is a partial perspective view of the air
duct outlet of **Fig. 1** with the central planar wall and
30 convexly-curved walls illustrated.

Fig. 3 is a top view of **Fig. 2**, taken along
lines 3-3.

Figs. 4-6 illustrate movement of the first vane
in the first air passageway of the air duct outlet of

Fig. 3 and the corresponding changes in air flow direction.

5 **Fig. 7** is a partial perspective view of the air duct outlet of **Fig. 1**, illustrating the internal manual-automatic assembly that permits manual operation and automatic oscillation of the vane within the first air passageway, according to embodiments of the present invention.

10 **Fig. 8** is a top section view of the air duct outlet of **Fig. 7**, taken along lines 8-8 and illustrating the actuator in the first position (manual direction control).

15 **Fig. 9** illustrates the air duct outlet of **Fig. 8** with the actuator in the second position (manual direction control).

Figs. 10-12 illustrates the air duct outlet of **Fig. 8** with the actuator in the third position (automatic oscillation).

20 **Fig. 13** is a front, perspective view of an air duct outlet, according to another embodiment of the present invention.

25 **Fig. 14** is a rear perspective view of the air duct outlet of **Fig. 13** illustrating the internal manual-automatic assembly that permits manual operation and automatic oscillation of a set of louvers.

DETAILED DESCRIPTION OF THE INVENTION

30 The present invention now is described more fully hereinafter with reference to the accompanying drawings, in which preferred embodiments of the invention are shown. This invention may, however, be embodied in many different forms and should not be construed as limited to the embodiments set forth herein; rather, these embodiments are provided so that this disclosure
35 will be thorough and complete, and will fully convey the

scope of the invention to those skilled in the art.

In the drawings, the thickness of lines, layers and regions may be exaggerated for clarity. It will be understood that when an element such as a layer, region, substrate, or panel is referred to as being "on" another element, it can be directly on the other element or intervening elements may also be present. In contrast, when an element is referred to as being "directly on" another element, there are no intervening elements present. It will be understood that when an element is referred to as being "connected" or "attached" to another element, it can be directly connected or attached to the other element or intervening elements may also be present. In contrast, when an element is referred to as being "directly connected" or "directly attached" to another element, there are no intervening elements present. The terms "upward", "downward", "vertical", "horizontal", and the like, are used herein for the purpose of explanation only.

Referring to **Figs. 1-12**, an air duct outlet **10** having manual/automatic air stream direction control, according to embodiments of the present invention, is illustrated. The illustrated air duct outlet **10** is designed for use within the interior compartments of vehicles, such as automobiles, trucks, trains, boats, aircraft, and the like. However, it is understood that the air duct outlet **10** may be utilized in various environments (e.g., in homes and offices) and is not limited only to use in vehicles.

The illustrated air duct outlet **10** includes a housing **12** having a front panel **13**, a top wall **14**, a bottom wall **16**, a central wall **18** and a pair of convexly-curved walls **20**, **22**, each positioned on respective opposite sides of the central wall **18**. The illustrated

central wall 18 has a generally planar configuration, but can have various shapes and configurations without limitation. Central wall 18 and convexly-curved wall 20 are in adjacent, spaced-apart relationship and define a first air passageway 24 (**Fig. 2**) having an inlet 24a and a flared outlet 24b, as illustrated. Similarly, central wall 18 and convexly-curved wall 22 are in adjacent, spaced-apart relationship and define a second air passageway 26 (**Fig. 2**) having an inlet 26a and a flared outlet 26b, as illustrated. As would be understood by those skilled in the art, the housing 12 is configured to be connected to a source of air flow, such as a fan, that supplies heated and/or cooled air (as well as ambient outside air) to the first and second air passageways 24, 26.

The illustrated air duct outlet 10 includes a plurality of fixed, generally horizontal louvers 29 (**Fig. 1**) that extend between convexly-curved walls 20, 22. Embodiments of the present invention may have various numbers and configurations of fixed louvers and are not limited to the illustrated louvers 29. Moreover, embodiments of the present invention need not have any fixed louvers.

Referring to **Fig. 3**, respective end portions 18a, 20a of the central wall 18 and convexly-curved wall 20 at the first air passageway inlet 24a are substantially parallel (*i.e.*, they generally extend along directions that are substantially parallel), and respective end portions 18b, 20b of the central wall 18 and convexly-curved wall 20 at the first air passageway outlet 24b are substantially orthogonal (*i.e.*, they generally extend along directions that are substantially perpendicular). Similarly, respective end portions 18a, 22a of the central wall 18 and convexly-curved wall 22 at the second air passageway inlet 26a are substantially

parallel, and respective end portions 18b, 22b of the central wall 18 and convexly-curved wall 22 at the second air passageway outlet 26b are substantially orthogonal.

In the illustrated embodiment, a first vane 30 is pivotally attached within the first air passageway 24 adjacent the first air passageway inlet 24a and is movable within a range of positions that are increasingly transverse to the central wall 18. Similarly, a second vane 32 is pivotally attached within the second air passageway 26 adjacent the second air passageway inlet 26a and is movable within a range of positions that are increasingly transverse to the central wall 18. Vanes 30, 32 may be pivotally attached within the respective air passageways 24, 26 in various ways (e.g., via pivot pins, hinges, etc.), without limitation. Vanes 30, 32 may be pivotally attached to the respective central wall 18 and/or to the top and/or bottom walls 14, 16 of the housing, as would be understood by one skilled in the art.

Air duct outlets according to embodiments of the present invention may be formed from various materials, without limitation. For example, the various components of the illustrated air duct outlet 10 may be formed from one or more types of polymeric materials, metals, wood, etc.

Operation of the first and second vanes 30, 32 to control the direction of air streams exiting from the respective first and second air passageways 24, 26 is identical. Accordingly, operation of only the first vane 30 is illustrated in Figs. 4-6 and described below.

In a first position (Fig. 4), the first vane 30 is disposed in face-to-face adjacent relationship with the central wall 18. In this position, an air stream flowing through the first air passageway 24 is not

deflected by the first vane 30 and the air stream exits from the outlet 24b in a direction A_1 that is substantially normal to the first air passageway outlet 24b (i.e., substantially parallel with the illustrated central wall 18). As the first vane 30 is pivoted from the first position (Fig. 4), an air stream flowing through the first air passageway 24 is deflected by an amount that increases with each position of the vane in the range. The air stream adheres to a portion of the convexly-curved wall 20 via a phenomenon known as the "Coanda effect" and by an amount that increases with each amount by which the air stream is deflected.

As known to those skilled in the art, the Coanda effect is the tendency for a moving fluid (either liquid or gas) to attach itself to a surface and flow along the surface. As a fluid moves across a surface a certain amount of friction occurs between the surface and the fluid, which tends to slow down the fluid as it moves across the surface. This resistance to flow tends to pull the fluid towards the surface, making it adhere to the surface, even as it bends around corners.

The more the air stream adheres to the convexly-curved wall 20, the greater the angle the exiting air stream has relative to the central wall 18. For example, in Fig. 5, the first vane 30 is moved partially within its range of movement. At the illustrated position, the air stream adheres to the convexly-curved wall 20 up until location P_1 resulting in the air stream exiting from the outlet 24b in a direction A_2 that is transverse to the central wall. In the illustrated embodiment, A_2 is at an angle of about 45° relative to the central wall 18. In Fig. 6, the first vane 30 is moved further within its range of movement. At the illustrated position, the air stream adheres to the convexly-curved wall 20 up until location P_2 resulting in

the air stream exiting from the outlet **24b** in a direction **A₃** that is substantially orthogonal (e.g., about 90°) to the central wall **18**.

Accordingly, an air stream flowing through the first air passageway **24** exits from the outlet **24b** in a direction that is increasingly transverse to a direction normal to the outlet **24b** as the first vane **30** is moved along its range of movement. Similarly, an air stream flowing through the second air passageway **26** would exit from the outlet **26b** in a direction that is increasingly transverse to a direction normal to the outlet **26b** as the second vane **32** is moved along its range of movement.

In the illustrated embodiment, each vane **30**, **32** is movable within a range of positions that can cause an air stream to have an exit direction from the first and second air passageway outlets **24b**, **26b** of between about 0° and about 90° relative to a direction normal to each respective outlet **24b**, **26b**. Moreover, because there are no movable louvers to reduce the effective areas of the outlets **24b**, **26b** (as there are with conventional air duct outlets), large pressure drops and lower flow rates are avoided even when an air stream exits at a 90° angle.

Referring back to **Fig. 1**, a pair of actuators **40a**, **40b** extend from the housing front panel **13** and are each operably coupled with a respective one of the first and second vanes **30**, **32** to allow a user to manually adjust air flow direction and to select automatic oscillation of air flow. A user desiring to manually adjust the direction of an air stream exiting from the first air passageway outlet **24b** moves actuator **40a** within a range of positions that are between first and second positions (**A** and **B**, **Fig. 7**). Movement of the actuator **40a** between the first and second positions causes the first vane **30** to be moved within its range of movement. A user desiring automatic oscillation of an air stream exiting

from the first air passageway outlet **24b** moves actuator **40a** to a third position (**C**, **Fig. 7**).

Similarly, a user desiring to manually adjust the direction of an air stream exiting from the second air passageway outlet **26b** moves actuator **40b** within a range of positions that are between the first and second positions **A**, **B**. Movement of the actuator **40b** between the first and second positions causes the first vane **32** to be moved within its range of movement. A user desiring automatic oscillation of an air stream exiting from the second air passageway outlet **26b** moves actuator **40b** to the third position **C**.

In the illustrated embodiment, the third position (automatic oscillation) is to the right of the first and second positions (manual direction control) for actuator **40a** and to the left of the first and second positions for actuator **40b**. However, embodiments of the present invention are not limited to any particular arrangement of an actuator with respect to automatic oscillation and manual direction control. Moreover, actuators **40a**, **40b** may be located virtually anywhere on the front panel **13**.

Referring now to **Figs. 7-12**, a manual-automatic assembly **50** that connects actuator **40a** and air-directing vane **30** and that permits manual operation and automatic oscillation, according to embodiments of the present invention, is illustrated. The illustrated assembly **50** includes an elongated reciprocating member **52** that is movably secured to the housing **12**. The reciprocating member **52** is configured for reciprocal movement along a longitudinal axis, indicated by **D₁**. A pin **53** extends downwardly from the reciprocating member **52**, as illustrated.

A drive source **54** is operably connected to the reciprocating member **52** and transmits reciprocating

motion to the reciprocating member 52. The illustrated drive source 54 may include an electrical motor that provides rotational motion via a drive shaft, as would be understood by those skilled in the art. A pin 60 is secured to a peripheral region of the drive source and engages a slot 58 in the reciprocating member 58, as illustrated.

Electrical motors and drive shafts extending therefrom are well understood by those skilled in the art and need not be described further herein. Various types and sizes of electrical motors may be utilized in accordance with embodiments of the present invention. Power for an electrical motor may be provided via various sources. For example, power may be provided from one or more batteries or from the electrical system of a vehicle. Ways of providing power to an electrical motor are within the scope of knowledge of those skilled in the art and need not be described further herein.

A slot 58 formed in a portion of the reciprocating member 52 receives the pin 60 and serves as a linkage to operably connect the rotating drive source 54 to the reciprocating member 52. The rotational motion of the pin is converted into reciprocating motion of the reciprocating member 52. The reciprocating member 52 may reciprocate continuously whether the actuator 40a is in position C (automatic oscillation) or between positions A and B (manual operation). Alternatively, the drive source 54 may only rotate when the actuator 40a is moved to position C, such that the reciprocating member 52 only reciprocates when the actuator 40a is moved to position C.

The actuator 40a, which extends through the housing 12 via slot 64 is connected to slide member 66, as illustrated. Actuator 40a and slide member 66 are slidably movable along a longitudinal axis indicated as D₂

between first, second and third positions, indicated by A, B, C, respectively. An elongated drive arm 68 is pivotally connected to the slide member 66 at a first end 68a and slidably engages a first linkage member 70 at a second end 68b. The illustrated first linkage member 70 is pivotally connected to the housing 12. A second linkage member 72 has an elongated configuration with opposite ends 72a, 72b. End 72a has a pinned connection with the first linkage member 70 and opposite end 72b has a pinned connection with a third linkage member 73. The third linkage member 73 is connected to the air-directing vane 30. A biasing member 76 is secured at a first end 76a to the housing 12 and to the third linkage member 73 at an opposite second end 76b. Biasing member 76 is configured to continuously urge the vane 30 to a substantially transverse direction relative to central wall 18. Biasing member 76 may be virtually any type of spring or elastic member.

Manual control of the air-directing vane 30 is performed when the actuator 40a is moved between first and second positions A, B. In Fig. 8, the actuator 40a is in the first position A which causes the air-directing vane 30 to be disposed in face-to-face adjacent relationship (Fig. 4) with the central wall 18. In this position, an air stream flowing through the first air passageway 24 is not deflected by the first vane 30 and the air stream exits from the outlet 24b in a direction A₁ that is substantially normal to the first air passageway outlet 24b (i.e., substantially parallel with the illustrated central wall 18).

In Fig. 9, the actuator 40a is in the second position B which causes the air-directing vane 30 to be in its transverse-most position relative to the central wall 18 (Fig. 6) such that the air stream exits from the

outlet 24b in a direction A₃ that is substantially orthogonal (e.g., about 90°) to the central wall 18. User movement of actuator 40a between the first and second positions A, B allows a user to manually control the direction of air flow exiting from the outlet 24b.

Accordingly, positions A, B represent endpoints of the range of manual movement of the air-directing vane 30.

Automatic oscillation of the air-directing vane 30 is initiated when the actuator 40a is moved to its third position C (Figs. 10-12). When the actuator 40a is moved to the third position C, the drive arm 68 is moved into position such that the reciprocating member pin 53 makes contact with the drive arm 68. The reciprocating motion of the reciprocating member 52 is transmitted to the drive arm 68 which pivots about end 68a. This pivotal movement causes the first linkage 70 to pivot about end 70a and thereby move the second and third linkages 72, 73, thereby moving the air-directing vane 30.

Biasing member 76 provides a constant pull on the third linkage 73 such that the vane 30 is constantly being pulled to the most transverse position. In conjunction with the reciprocating pin 53 engaging the first linkage member 70, the biasing member 76 causes the vane 30 to reciprocate between the position illustrated in Fig. 4 and the position illustrated in Fig. 6. When actuator 40a is in the second position B, drive arm 68 is sufficiently to the right of reciprocating pin 53 such that reciprocating pin 53 cannot engage drive arm 68.

Figs. 10-12 illustrate a portion of a cycle of automatic oscillation of air-directing vane 30. In Fig. 10, pin 60 is at a "9 o'clock" position which causes the air-directing vane 30 to be in its most transverse orientation relative to the central wall 18. In Fig. 11 the drive shaft has rotated the pin 60 to the "12

o'clock" position and the air-directing vane 30 is moved to a less transverse orientation relative to the central wall 18. In Fig. 12, the drive shaft has rotated the pin 60 to the "3 o'clock" position and the air-directing vane 30 is moved to face-to-face orientation with the central wall 18.

Various ways of mechanically linking each actuator 40a, 40b with a respective vane 30, 32 may be utilized. Embodiments of the present invention are not limited to the illustrated assembly 50 and the various linkages and configurations thereof.

Referring now to Figs. 13-14, an air duct outlet 110, according to other embodiments of the present invention, is illustrated. The illustrated air duct outlet 110 includes a housing 112. The housing 112 includes a front wall 113 having an opening 114 therein that allows air to pass therethrough. A thumbwheel 116 extends through the front wall 114, as illustrated, and is utilized to adjust air flow in an up and down direction as described below. As would be understood by those skilled in the art, the air duct outlet 110 is connected to a source of air flow, such as a fan, to supply heated and/or cooled air (as well as ambient outside air) to an interior compartment of a vehicle. Air is supplied into the rear of the housing 112 through an opening 111 formed by the housing walls 112a-112d and exits through the first opening 114 in the front of the housing 112.

The illustrated air duct outlet 110 includes a first set of elongated, horizontal louvers 120 that are in generally parallel, spaced-apart relationship with each other. The first set of louvers 120 are pivotally mounted within the air duct outlet 110 and pivotal movement thereof is controlled by thumbwheel 116, as would be understood by those skilled in the art. A second

set of elongated louvers 130 are in generally parallel,
 spaced-apart relationship with each other. The second set
 of louvers 130 are pivotally mounted within the air duct
 outlet and pivotal movement thereof is controlled by the
 5 manual-automatic air stream direction control assembly
 50, described above with respect to Figs. 1-12. The
 assembly 50, is connected to a single one of the vertical
 louvers 130. Because the vertical louvers 130 are
 pivotally connected together, movement of one louver
 10 causes all to move, as would be understood by those
 skilled in the art.

Manual control of the air-directing louvers 130
 occurs when the actuator 140 is moved between first and
 second positions A, B., as described above. Automatic
 15 oscillation of the louvers 130 occurs when the actuator
 is moved to position C.

The foregoing is illustrative of the present
 invention and is not to be construed as limiting thereof.
 Although a few exemplary embodiments of this invention
 20 have been described, those skilled in the art will
 readily appreciate that many modifications are possible
 in the exemplary embodiments without materially departing
 from the novel teachings and advantages of this
 invention. Accordingly, all such modifications are
 25 intended to be included within the scope of this
 invention as defined in the claims. Therefore, it is to
 be understood that the foregoing is illustrative of the
 present invention and is not to be construed as limited
 to the specific embodiments disclosed, and that
 30 modifications to the disclosed embodiments, as well as
 other embodiments, are intended to be included within the
 scope of the appended claims. The invention is defined by
 the following claims, with equivalents of the claims to
 be included therein.

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